6. Bearing tolerances

6.1 Dimensional and rotational accuracy

Bearing "tolerances" or dimensional accuracy and running accuracy, are regulated by ISO and JIS standards, JIS B 1514 (rolling bearing tolerances) series. For dimensional accuracy, these standards prescribe the tolerances necessary when installing bearings on shafts or in housings. Running accuracy is defined as the allowable limits for bearing runout during operation.

Dimensional accuracy

Dimensional accuracy constitutes the acceptable values for bore diameter, outside diameter, assembled bearing width, and bore diameter uniformity as seen in chamfer dimensions, allowable inner ring tapered bore deviation and shape error. Also included are variation of mean bore diameter within a plane, outside diameter within a plane, mean outside diameter within a plane, as well as raceway thickness (for thrust bearings).

Running accuracy

Running accuracy constitutes the acceptable values for inner and outer ring radial runout and axial runout, inner ring side surface squareness, and outer ring outside diameter squareness.

Allowable rolling bearing tolerances have been established according to precision classes. Bearing precision is stipulated as JIS Class 6, Class 5, Class 4, or Class 2, with precision rising from ordinary precision indicated by JIS Class 0.

Table 6.1 indicates which standards and precision classes are applicable to the major bearing types. Table 6.2 shows a relative comparison between JIS B 1514 precision class standards and other standards.

For details of allowable limitations and values, refer to Table 6.4 through Table 6.10, which are described in the application table column of Table 6.1. Allowable values for chamfer dimensions are shown in Table 6.11. Allowable limitations and values for radial bearing inner ring tapered bores are shown in Table 6.12.

Table 6.1 Bearing types and applicable tolerance

	Bearing type	Applicable standard		Acc	uracy cla	ss		Tolerance table
Deep groo	ove ball bearings		Class 0	Class 6	Class 5	Class 4	Class 2	
Angular c	ontact ball bearings		Class 0	Class 6	Class 5	Class 4	Class 2	
Self-align	ing ball bearings	JIS B 1514-1	Class 0	_	_	_	_	Table 6.4
Cylindrica	l roller bearings	(ISO 492)	Class 0	Class 6	Class 5	Class 4	Class 2	Table 6.4
Needle ro	ller bearings		Class 0	Class 6	Class 5	Class 4	_	
Spherical	roller bearings		Class 0	_	_	_	_	
	Metric series (single-row)	JIS B 1514	Class 0, 6X	Class 6 ¹⁾	Class 5	Class 4	_	Table 6.5
Tapered roller	Metric series (double-row/four-row)	BAS 1002	Class 0	_	_	_	_	Table 6.7
bearings	Inch series	ANSI/ABMA Std.19	Class 4	Class 2	Class 3	Class 0	Class 00	Table 6.6
	J series	ANSI/ABMA Std.19.1	Class K	Class N	Class C	Class B	Class A	Table 6.8
Thrust ba	II bearings	JIS B 1514-2	Class 0	Class 6	Class 5	Class 4	_	Table 6.9
Thrust sp	herical roller bearings	(ISO 199)	Class 0	_	_	_	_	Table 6.10

¹⁾ The class is the NTN standard class.

Table 6.2 Comparison of tolerance classifications of national standards

Standard	Applicable standard		Acc	uracy cla	ass		Bearing type
Japanese industrial	JIS B 1514-1	Class 0, 6	Class 6	Class 5	Class 4	Class 2	Radial bearings
standards (JIS)	JIS B 1514-2	Class 0	Class 6	Class 5	Class 4	_	Thrust bearings
	ISO 492	Normal class Class 6X	Class 6	Class 5	Class 4	Class 2	Radial bearings
International Organization for	ISO 199	Normal class	Class 6	Class 5	Class 4	_	Thrust bearings
Standardization (ISO)	ISO 578	Class 4	_	Class 3	Class 0	Class 00	Tapered roller bearings (Inch series)
	ISO 1224	_	_	Class 5A	Class 4A	_	Precision instrument bearings
Deutsches Institut fur Normung (DIN)	DIN 620	P0	P6	P5	P4	P2	All types
American National	ANSI/ABMA Std.20 ¹⁾	ABEC-1 RBEC-1		ABEC-5 RBEC-5	ABEC-7	ABEC-9	Radial bearings (excluding tapered roller bearings)
Standards Institute (ANSI) American Bearing Manufacturer's	ANSI/ABMA Std.19.1	Class K	Class N	Class C	Class B	Class A	Tapered roller bearings (Metric series)
Association (ABMA)	ANSI/ABMA Std.19	Class 4	Class 2	Class 3	Class 0	Class 00	Tapered roller bearings (Inch series)

^{1) &}quot;ABEC" is applied to ball bearings and "RBEC" to roller bearings.

Bearing Tolerances

Note: 1. JIS B 1514 series, ISO 492, 199, and DIN 620 have the same specification level.

^{2.} The tolerance and allowance of JIS B 1514 series are slightly different from those of ABMA standards.

Ordinary precision JIS Class 0 is applied to general roller bearings. However, depending on the conditions and applications, bearings with JIS Class 5 or higher may be necessary.

Table 6.3 shows application examples of accuracy class according to the required performance of bearings to be used.

Table 6.3 Application example of accuracy class

Required performance	Application example	Applied accuracy class
Accuracy under high speed	Machine tool main spindles Printing machine body bearings Magnetic tape guides	JIS Class 5, JIS Class 4 or higher JIS Class 5 JIS Class 5
High rotational speed	Jet engine main spindles Turbochargers Machine tool main spindles Touchdown bearings of magnetic bearing spindles for turbo- molecular pumps	JIS Class 4 or higher Equivalent to JIS Class 4 JIS Class 5, JIS Class 4 or higher JIS Class 5
Low torque low noise	Machine tool main spindles Hubs of road bikes Cleaner motors Hand spinners Fan motors	JIS Class 5, JIS Class 4 or higher JIS Class 5 JIS Class 0 JIS Class 0 JIS Class 0

6.2 JIS terms

The following is a description of JIS accuracy terms used in **Table 6.4**.

(However, the outside diameter surface is omitted because the meaning is similar.)

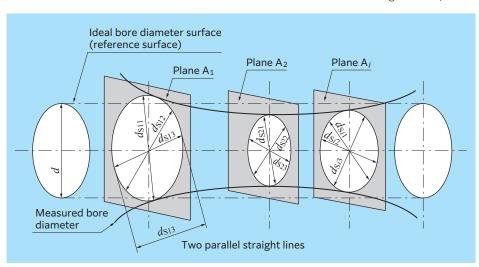


Fig. 6.1 Shape model figure

Terms	Quantifiers	Description
ICITIS	Quantificis	·
Nominal bore diameter	d	Reference dimension representing the bore diameter size, and reference value with respect to the dimensional difference of the actual bore diameter surface.
Single bore diameter	d_{s}	Distance between two parallel straight lines that are in contact with the intersection line of the actual bearing bore diameter surface and the radial plane.
Deviation of a single bore diameter	$\Delta_{d\mathrm{s}}$	Difference between $d_{\rm S}$ and d (difference of nominal diameter serving as the measured bore and standard).
Mean bore diameter in a single plane	$d_{ m mp}$	Arithmetic mean of the maximum and minimum measured bore diameters within one radial plane. In the model figure, in arbitrary radial plane A_{ii} , when the maximum bore diameter is d_{si1} and the minimum bore diameter is d_{si3} , the value is obtained by $(d_{si1} + d_{si3})/2$. There is one value for each plane.
Mean bore diameter	$d_{ m m}$	Arithmetic mean of the maximum and minimum measured bore diameters obtained from all the cylindrical surfaces. In the model figure, when the maximum measured bore diameter is d_{s11} and the minimum measured bore diameter is d_{s23} , which are obtained from the all the planes $A_1, A_2, \dots A_i$, the mean bore diameter is obtained by $(d_{s11} + d_{s23})/2$. There is one value for one cylindrical surface.
Deviation of mean bore diameter	$\Delta_{d\mathrm{m}}$	Difference between the mean bore diameter and the nominal bore diameter.
Deviation of mean bore diameter in a single plane	$\Delta_{d\mathrm{mp}}$	Difference between the arithmetic mean and the nominal bore diameter of the maximum and minimum measured bore diameters within one radial plane. The value is specified in JIS.
Variation of bore diameter in a single plane	$V_{d\mathrm{sp}}$	Difference between the maximum and minimum measured bore diameters within one radial plane. In the model figure, in radial plane A_1 , when the maximum measured bore diameter is d_{s11} and the minimum measured bore diameter is d_{s13} , the difference is V_{dsp} and one value can be obtained for one plane. This characteristic is an index that indicates the roundness. The value is specified in JIS.
Variation of mean bore diameter	$V_{d\mathrm{mp}}$	Difference between the maximum and minimum values of the mean bore diameter within a plane that are obtained from all the planes. A unique value is obtained for each product, and it is near to cylindricity (that is different from geometric cylindricity). The value is specified in JIS.
Nominal inner ring width	В	Distance between both theoretical side surfaces of a raceway. This value is a reference dimension that represents the raceway surface (distance between both side surfaces).
Single inner ring width	B_{s}	Distance between two intersections. The straight is perpendicular to the plane that is in contact with the inner ring reference side and both actual side surfaces. This value represents the actual width dimension of an inner ring.
Deviation of a single inner ring width	$\Delta_{B_{ m S}}$	Difference between the measured inner ring width and the nominal inner ring width. This value is also the difference between the measured inner ring width dimension and the reference dimension that represents the inner ring width. The value is specified in JIS.
Variation of inner ring width	V_{Bs}	Difference between the maximum and minimum measured inner ring widths, which are specified in JIS.
Radial runout of inner ring of assembled bearing	K _{ia}	Difference between the maximum and minimum values of the radial distance between the inner ring bore diameter at each angle position and one fixed point of the outer ring outside diameter surface with respect to radial runout.
Axial runout of inner ring of assembled bearing	$S_{ m ia}$	Difference between the maximum and minimum values of the axial distance between the inner ring reference side surface at each angle position and one fixed point of the outer ring outside diameter surface with respect to half the radial distance of the raceway contact diameter from the inner ring central axis and the inner ring of a deep groove ball bearing.

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Table 6.4 Tolerance of radial bearings (except tapered roller bearings) Table 6.4 (1) Inner rings

bo	ninal ore neter	[Devia	tion			bore plan		mete	er ir	n a			V	aria	tion	of b	ore o	diam	ete	r in a	sing	le pl	ane		
	d d					Δ	<i>l</i> mp												V_{d}	sp						
m Over	ım Incl.		ass O r Lower								ss 2 ¹⁾ r Lower	Class 0			Class 4			Class 6		Class 4	O, 1 4 Class 2			r serie Class 5 Max.	Class 4	3, 4 4 Class 2
0.6 2.5 10		0 0 0	-8 -8 -8	0 0	-7 -7 -7	0 0	-5 -5 -5	0 0 0	-4 -4 -4	0 0 0	-2.5 -2.5 -2.5	10 10 10	9 9 9	5 5 5	4	2.5 2.5 2.5	8 8 8	7 7 7	4 4 4	3 3	2.5 2.5 2.5	6 6 6	5 5 5	4 4 4	3 3	2.5 2.5 2.5
18 30 50	30 50 80	0 0 0	-10 -12 -15	0 0 0	-8 -10 -12	0 0 0	-6 -8 -9	0 0 0	-5 -6 -7	0 0 0	-2.5 -2.5 -4	13 15 19	10 13 15	6 8 9	5 6 7	2.5 2.5 4	10 12 19	8 10 15	5 6 7	4 5 5	2.5 2.5 4	8 9 11	6 8 9	5 6 7	4 5 5	2.5 2.5 4
80 120 150	120 150 180	0 0 0	-20 -25 -25	0 0 0	-15 -18 -18	0 0 0	-10 -13 -13	0 0 0	-8 -10 -10	0 0 0	-5 -7 -7	25 31 31	19 23 23	10 13 13	8 10 10	5 7 7	25 31 31	19 23 23	8 10 10	6 8 8	5 7 7	15 19 19	11 14 14	8 10 10	6 8 8	5 7 7
180 250 315	250 315 400	0 0 0	-30 -35 -40	0 0 0	-22 -25 -30	0 0 0	-15 -18 -23	0 - -	-12 - -	0 - -	-8 - -	38 44 50	28 31 38	15 18 23	12 - -	8 _ _	38 44 50	28 31 38	12 14 18	9 - -	8 _ _	23 26 30	17 19 23	12 14 18	9 — —	8 _ _
400 500 630	500 630 800	0 0 0	-45 -50 -75	0 0 —	-35 -40 -	=	_ _ _	_ _ _	_ _ _	_ _ _	_ _ _	56 63 —	44 50 —	_ _ _	_ _ _	_ _ _	56 63 —	44 50 —	_ _ _	_ _ _	_ _ _	34 38 —	26 30 —	_ _ _	_	_ _ _
800 1 000 1 250 1 600	1 000 1 250 1 600 2 000	0 0 0	-100 -125 -160 -200	_ _ _	_ _ _	_ _ _	_ _ _	_ _ _	_ _ _	_ _ _	_ _ _	_ _ _ _	_ _ _	_ _ _	_ _ _	_ _ _	_ _ _ _	_ _ _	_ _ _	_ _ _	_ _ _ _	_ _ _ _	_ _ _	_ _ _	_ _ _	_ _ _

¹⁾ The dimensional difference Δ_{ds} of the measured bore diameter applied to Classes 4 and 2 is the same as the tolerance of dimensional difference Δ_{dmp} of the mean bore diameter within a plane. However, the dimensional difference is applied to diameter series 0, 1, 2, 3 and 4 for Class 4, and also to all the diameter series for Class 2.

Table 6.4 (2) Outer rings

Nom outs diam	side	De	eviatio	on c			utsid plan		iame	ter	in a			Vari	atio	n of	outs	ide	dian	nete	er in a	a sing	gle pl	lane ⁶	i)	
m Over	m Incl.		ass O r Lower		ass 6 r Lower	Cla	Omp iss 5 r Lower l						Class 6	ter s Class 5 Max.	Class 4			mete Class 6	V_D en b er se Class 5 Max.	eari ries Class 4	0, 1		Class 6	r serie Class 5 Max.	Class 4	
2.5 6 18	8) 6 18 30	0 0 0	-8 -8 -9	0 0	-7 -7 -8	0 0 0	-5 -5 -6	0 0 0	-4 -4 -5	0 0 0	-2.5 -2.5 -4	10 10 12	9 9 10	5 5 6	4 4 5	2.5 2.5 4	8 8 9	7 7 8	4 4 5	3 3 4	2.5 2.5 4	6 6 7	5 5 6	4 4 5	3 3 4	2.5 2.5 4
30	50	0	-11	0	-9	0	-7	0	-6	0	-4	14	11	7	6	4	11	9	5	5	4	8	7	5	5	4
50	80	0	-13	0	-11	0	-9	0	-7	0	-4	16	14	9	7	4	13	11	7	5	4	10	8	7	5	4
80	120	0	-15	0	-13	0	-10	0	-8	0	-5	19	16	10	8	5	19	16	8	6	5	11	10	8	6	5
120	150	0	-18	0	-15	0	-11	0	-9	0	-5	23	19	11	9	5	23	19	8	7	5	14	11	8	7	5
150	180	0	-25	0	-18	0	-13	0	-10	0	-7	31	23	13	10	7	31	23	10	8	7	19	14	10	8	7
180	250	0	-30	0	-20	0	-15	0	-11	0	-8	38	25	15	11	8	38	25	11	8	8	23	15	11	8	8
250	315	0	-35	0	-25	0	-18	0	-13	0	-8	44	31	18	13	8	44	31	14	10	8	26	19	14	10	8
315	400	0	-40	0	-28	0	-20	0	-15	0	-10	50	35	20	15	10	50	35	15	11	10	30	21	15	11	10
400	500	0	-45	0	-33	0	-23	—	-	—	-	56	41	23	—	—	56	41	17	—	—	34	25	17	—	—
500	630	0	-50	0	-38	0	-28	_	_	_	_	63	48	28	-	_	63	48	21	_	_	38	29	21	_	_
630	800	0	-75	0	-45	0	-35	_	_	_	_	94	56	35	-	_	94	56	26	_	_	55	34	26	_	_
800	1 000	0	-100	0	-60	—	-	_	_	_	_	125	75	—	-	_	125	75	—	_	_	75	45	—	_	_
1 250 1 600	1 250 1 600 2 000 2 500	0 0 0 0	-125 -160 -200 -250	_ _ _	- - -	- - -	- - -	_ _ _	_ _ _	_ _ _	- - -	1 1 1 1	_ _ _	_ _ _	- - -	_ _ _		_ _ _	_ _ _	_ _ _	- - -	_ _ _ _	_ _ _	_ _ _	_ _ _	

⁵⁾ The dimensional difference ΔD_{S} of the measured outside diameter applied to Classes 4 and 2 is the same as the tolerance of dimensional difference Δ_{Dmp} of the mean outside diameter within a plane. However, the dimensional difference is applied to diameter series 0, 1, 2, 3 and 4 for Class 4, and also to all the diameter series for Class 2.

Unit: μm

- 1	/ari me dia	an	bc	ore		(of fa	inr	ner en	rir ıbl		of in	ner rin resp	ularity ig face ect to	of i	nner ssen			eviati	ion	of a		J	nne	r ring	wi	dth		er r	atio	wic	
Class O	Class 6	V _d 1 S Class Ma	5 Cl		Class 2	Class 0		<i>1</i>	ari K _{ia} Ilass 5 Max	Class	4 Class 2		he bo S _d Class 4 Max	4 Class 2	Class 5	eari Sia Class Max	2) 4 Class 2		0 Class 6	Class		ngs ()	lass 2	Class (plex be Class 6 or Lower	Class 5	Class 4	Class 0	Class 6	$V_{B{ m S}}$ Class 5 Max.	Class 4	Class 2
6 6 6	5 5 5	3	3 2)	1.5 1.5 1.5	10 10 10		5 6 7	4	2.5	1.5 1.5 1.5	7 7 7	3 3 3	1.5 1.5 1.5	7 7 7	3 3 3	1.5 1.5 1.5		-40 -120 -120	Ō	-40 -40 -80	Ō	-40 -40 -80	0	-250 -250	0	-250 -250 -250			5	2.5 2.5 2.5	1.5
8 9 11	6 8 9	4	1 3		1.5 1.5 2	13 15 20	1		4 5 5	4	2.5 2.5 2.5	8 8 8	4 4 5	1.5 1.5 1.5	8 8 8	4 4 5	2.5 2.5 2.5	0	-120 -120 -150	0	-120	0	-120	0	-250 -250 -380	0		20				1.5 1.5 1.5
15 19 19	14	-	5 4 7 5 7 5	5	2.5 3.5 3.5	25 30 30	1	8	6 8 8	6	2.5 2.5 5	9 10 10	5 6 6	2.5 2.5 4	9 10 10	5 7 7	2.5 2.5 5	0	-200 -250 -250	Ō	-250	Ō	-250	0	-380 -500 -500	0		30	25 30 30	7 8 8	5	2.5 2.5 4
23 26 30	19				4 — —	40 50 60	2	5	10 13 15	8 - -	5 - -	11 13 15	7 _ _	5 _ _	13 15 20	8 - -	5 _ _	0 0 0	-300 -350 -400	Ō	-300 - -	0 -	-300 - -	0	-500 -500 -630	0	-500 - -	35	30 35 40	10 13 15	6 - -	5 — —
34 38 55	30	-		_	_	65 70 80	4		_ _ _	_ _ _	_ _ _	_ _ _	_ _ _	_ _ _	 - -	_ _ _	_ _ _	0 0 0	-450 -500 -		_ _ _	_ _ _	_ _ _	_ _ _	- - -	_ _ _	_ _ _	50 60 70		_ _ _	_ _ _	_ _ _
75 94 120 150	_ _ _	-		- - -	_ _ _	90 100 120 140	-	-	_ _ _ _	_ _ _	_ _ _	_ _ _ _	_ _ _	_ _ _	_ _ _ _	_ _ _	_ _ _	0 0 0 0	_ _ _	_ _ _	_ _ _	_ _ _		_ _ _ _		_ _ _ _	_ _ _	80 100 120 140	_	_ _ _ _	_ _ _	_ _ _ _

²⁾ Applies to ball bearings such as deep groove ball bearings and angular ball bearings. 4) The nominal bore diameter of bearings of 0.6 mm is included in this dimensional division. 3) Applies to individual raceway rings manufactured for combined bearing use.

Unit: µm

diamete plane Sealed/sh dia Diame	n of outside r in a single $v_{Dsp}^{(6)}$ ield bearings meter ter series				tsid ter		c	of ou f as be	al ruuter semearir K_{ea}	ring ble	5	of o outs with	endicu outer r ide su respe he fac $S_{ m D}$	ring rface ect to	of r ass b	al rur f out ing c emb earir Sea	of led	Deviation of a single outer ring width $\Delta C_{\rm S}$	Variation ring v	vidth
2, 3, 4 Class 0 N	0,1,2,3,4 Class 6 Max.	Class 0		Class 5 Max.		4 Class 2	Class 0		Class 5 Max.		Class 2	Class 5	Class 4 Max.			Class 4 Max.		All classes	Class 0,6 Class 5	
10 10 12	9 9 10	6 6 7	5 5 6	3	2 2 2.5	1.5 1.5 2	15 15 15	8 8 9	5 5 6	3	1.5 1.5 2.5	8 8 8	4 4 4	1.5 1.5 1.5	8 8 8	5 5 5	1.5 1.5 2.5	Depends on tolerance	Depends on	5 2.5 1.5 5 2.5 1.5 5 2.5 1.5
16 20 26	13 16 20	8 10 11	7 8 10	4 5 5	3 3.5 4	2 2 2.5	20 25 35	10 13 18	7 8 10		2.5 4 5	8 8 9	4 4 5	1.5 1.5 2.5	8 10 11	5 5 6	2.5 4 5	of Δ_{Bs} in relation to d of the same	of V_{Bs} in relation to d of the same	8 / 25
30 38 —	25 30 —	14 19 23	11 14 15	7	5 5 6	2.5 3.5 4	40 45 50	20 23 25	11 13 15		5 5 7	10 10 11	5 5 7	2.5 2.5 4	13 14 15	7 8 10	5 5 7	bearing	bearing	8 5 2.5 8 5 2.5 10 7 4
- - -	_ _ _	26 30 34	19 21 25	9 10 12	7 8 —	4 5 —	60 70 80	30 35 40	18 20 23	11 13 -		13 13 15	8 10 —	5 7 —	18 20 23	10 13 —	7 8 —			11 7 5 13 8 7 15 — —
_ _ _	=	38 55 75	29 34 45	14 18 —	_ _ _	_ _ _	100 120 140	50 60 75	25 30 —	_ _ _	_	18 20 —	_ _ _	_ _ _	25 30 —	_	_			18 20
_ _ _	_	_ _ _	=	_	=	=	160 190 220	_	_	=	_	_ _ _	=	_	_ _ _	_	=			
_	_	_	_	_	_	_	250	-	_	_	_	-	_	_	_	_	_			

⁸⁾ The nominal bore diameter of bearings of 2.5 mm is included in this dimensional division. 6) Applies to cases where snap rings are not installed on the bearings. 7) Applies to ball bearings such as deep groove ball bearings and angula r ball bearings

Table 6.5 Tolerance of tapered roller bearings (metric series) Table 6.5 (1) Inner rings

Nom bo diam	re		Devia bore si	dia		r in			iatior eter pla				iation ore dia			of	adial inner mble	ring	of	of inner with res	
a	l				dmp .				V_a	<i>l</i> sp			V_d	mp			K	ia		the	
			ss 0		ıss 6 ¹		2)	Class 0				Class 0				Class 0					d
m Over			ss 6× r Lower I					Class 6×	Class 6 ^{1,} Ma		Class 4	Class 6×	Class 6 1) Ma		Class 4	Class 6x	Class 6 1) Ma		Class 4		Class 4 ax.
10 18	18 30	0	-12 -12	0	-7 -8	0	-5 -6	12 12	7 8	5 6	4	9	5 6	5 5	4	15 18	7 8	5 5	3	7 8	3 4
30	50	0	-12	-	-10		-8	12	10	8	6	9	8	5	5	20	10	6	4	8	4
50	80 120	0	-15 -20	0	-12	_	-9 -10	15 20	12 15	9 11	7 8	11 15	9 11	6 8	5 5	25 30	10 13	7 8	4 5	8	5 5
80 120	180	0	-25	0	-15 -18	-	-13	25	18	14	10	19	14	9	7	35	18	11	6	9 10	6
180	250	0	-30	0	-22	0	-15	30	22	17	11	23	16	11	8	50	20	13	8	11	7
250 315	315 400	0	-35 -40	_	_	_	_	35 40	_	_	_	26 30	_	_	_	60 70	_	_	_	_	_

Table 6.5 (2) Outer rings

Nom outs diam	ide	Devia outsid si	e dia		er i			neter	of out in a si ane		1	iation side c			of	outer	runou ring d bea	of	of out outside	licularity er ring surface
L)		Δ_L	Omp				V_L	Osp			V_D	mp			K	ea		with res	spect to face
mı Over		Class 0 Class 6× Upper Lower	Cla		Clas	ss 4 ⁴⁾	Class 0 Class 6×) Class 5 ax.	Class 4	Class 0 Class 6×	Class 6 ¹⁾ Ma		Class 4	Class 0 Class 6×	Class 6 ¹⁾ Ma		Class 4	S Class 5 Ma	
18 30 50	30 50 80	0 -12 0 -14 0 -16		-8 -9 -11	-	-6 -7 -9	12 14 16	8 9 11	6 7 8	5 5 7	9 11 12	6 7 8	5 5 6	4 5 5	18 20 25	9 10 13	6 7 8	4 5 5	8 8 8	4 4 4
80 120 150	120 150 180	0 -18 0 -20 0 -25	0	-13 -15 -18	0	-10 -11 -13	18 20 25	13 15 18	10 11 14	8 8 10	14 15 19	10 11 14	7 8 9	5 6 7	35 40 45	18 20 23	10 11 13	6 7 8	9 10 10	5 5 5
180 250 315	250 315 400	0 -30 0 -35 0 -40	0	-20 -25 -28	0	-15 -18 -20	30 35 40	20 25 28	15 19 22	11 14 15	23 26 30	15 19 21	10 13 14	8 9 10	50 60 70	25 30 35	15 18 20	10 11 13	11 13 13	7 8 10
400 500	500 630	0 -45 0 -50	_	_	_	_	45 60	_	_	_	34 38	_	_	_	80 100	_	_	_	_ _	_

Unit: μ m

Axial runout of inner ring of assembled	Dev	viation o	of a sing	gle innei	ring w	idth	De	eviation	of the a	actual a g width		ed
bearing			Δ	<i>B</i> s					Δ	Ts		
S_{ia}	Cla	ss 0			Cla	ss 5	Cla	ss 0			Cla	ss 5
Class 4	Cla	ss 6	Clas	s 6×	Cla	ss 4	Cla	ss 6	Clas	s 6×	Cla	ss 4
Max.	Upper	Lower	Upper	Lower	Upper	Lower	Upper	Lower	Upper	Lower	Upper	Lower
3	0	-120	0	-50	0	-200	+200	0	+100	0	+200	-200
4	0	-120	0	-50	0	-200	+200	0	+100	0	+200	-200
4	0	-120	0	-50	0	-240	+200	0	+100	0	+200	-200
4	0	-150	0	-50	0	-300	+200	0	+100	0	+200	-200
5	0	-200	0	-50	0	-400	+200	-200	+100	0	+200	-200
7	0	-250	0	-50	0	-500	+350	-250	+150	0	+350	-250
8	0	-300	0	-50	0	-600	+350	-250	+150	0	+350	-250
_	0	-350	0	-50	_	_	+350	-250	+200	0	_	_
_	0	-400	0	-50	_	_	+400	-400	+200	0	_	

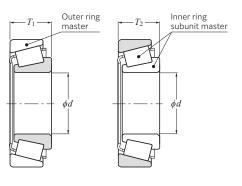
Table 6.5 (3) Effective width of inner subunits and outer rings Unit: μ m

Unit: μ m

$ \begin{array}{c} \textbf{Axial runout} \\ \textbf{of outer ring} \\ \textbf{of assembled} \\ \textbf{bearing} \\ S_{ea} \\ \hline \textbf{Class 4} \\ \textbf{Max.} \end{array} $	Deviation of a ring w Δ C Class 0, Class 6¹ Class 5, Class 4 Upper Lower	idth 's 'Class	s 6× ⁵⁾
5 5 5 6 7 8 10 10 13	Depends on tolerance of Δ_{Bs} in relation to d of the same bearing	0 0 0 0 0 0	-100 -100 -100 -100 -100 -100 -100 -100

⁵⁾ Applies to bearings with a nominal bore diameter *d* over 10 mm and 400 mm or less.

Nom bo diam	re leter	effec	tive wi	f the addth of itembled outer ri	nner with	effec ring	ation o tive wi assem ster inn Δ	dth of object when sub- bled when sub-	outer ith a
m Over	m Incl.			Clas Upper			ss 0 Lower		
10 18 30	18 30 50	+100 +100 +100	0 0 0	+50 +50 +50	0 0 0	+100 +100 +100	0 0 0	+50 +50 +50	0 0 0
50 80 120	80 120 180		0 -100 -150	+50 +50 +50	0 0 0	+100 +100 +200	0 -100 -100	+50 +50 +100	0 0 0
180 250 315	250 315 400	+150 +150 +200	-150	+50 +100 +100	0 0 0	+200	-100 -100 -200	+100 +100 +100	0 0 0



¹⁾ Class 6 is the **NTN** standard class.
2) The dimensional difference Δ_{ds} of the measured bore diameter applied to Class 4 is the same as the tolerance of dimensional difference Δ_{dmp} of the mean bore diameter within a plane.

 ³⁾ Does not apply to bearings with flange.
 4) The dimensional difference Δ_{Ds} of the measured outside diameter applied to Class 4 is the same as the tolerance of dimensional difference Δ_{Dmp} of the mean outside diameter within a plane.

Table 6.6 (1) Inner rings

Unit: μ m

Nominal bor	e diameter				Deviatio	n of a sin	gle bore	diameter			
a	!					Δ	ds				
mm (i		Cla		Cla			ss 3	Cla		Clas	
Over	Incl.	Upper	Lower	Upper	Lower	Upper	Lower	Upper	Lower	Upper	Lower
	76.2 (3)	+13	0	+13	0	+13	0	+13	0	+8	0
76.2 (3)	266.7 (10.5)	+25	0	+25	0	+13	0	+13	0	+8	0
266.7 (10.5)	304.8 (12)	+25	0	+25	0	+13	0	+13	0	_	_
304.8 (12)	609.6 (24)	+51	0	+51	0	+25	0	_	_	_	_
609.6 (24)	914.4 (36)	+76	0	_	_	+38	0	_	_	_	_
914.4 (36)	1 219.2 (48)	+102	0	_	_	+51	0	_	_	_	_
1 219.2 (48)		+127	0	_	_	+76	0	_	_	_	

Table 6.6 (2) Outer rings

Unit: μ m

Nominal outs	ide diameter			D	eviation	of a sing	e outside	e diamete	er		
I)					Δ	<i>D</i> s				
mm (inch)	Cla	ss 4	Cla	ss 2	Cla	ss 3	Cla	ss 0	Clas	s 00
Over	Incl.	Upper	Lower	Upper	Lower	Upper	Lower	Upper	Lower	Upper	Lower
	266.7 (10.5)	+25	0	+25	0	+13	0	+13	0	+8	0
266.7 (10.5)	304.8 (12)	+25	0	+25	0	+13	0	+13	0	_	_
304.8 (12)	609.6 (24)	+51	0	+51	0	+25	0	_	_	_	_
609.6 (24)	914.4 (36)	+76	0	+76	0	+38	0	_	_	_	_
914.4 (36)	1 219.2 (48)	+102	0	_	_	+51	0	_	_	_	_
1 219.2 (48)		+127	0			+76	0				

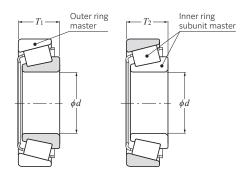
Table 6.6 (3) Assembly width of single-row bearings, combination width of 4-row bearings, effective width of inner ring subunits, effective width of outer rings

Nomin diam د	eter	dian	l outside neter D	D	eviatio	bearing	l assem g width $T_{ m S}$		ngle ro	w	Deviate four-row overall Δ_{B2s} ,	width
mm (Over			(inch) Incl.		ss 4 · Lower	ss 2 Lower		ss 3 Lower		0,00 Lower		1,2,3,0 Lower
 101.6 (4) 304.8 (12)	101.6 (4) 304.8 (12) 609.6 (24)	_	508.0 (20)		0 -254 -381	0	+203 +203 +203		+203 +203 —	-203 -203 -	+1 524 +1 524 +1 524	-1 524 -1 524 -1 524
304.8 (12) 609.6 (24)	609.6 (24) —	508.0 (20)	_		-381 -381		+381 +381		_	_	+1 524 +1 524	-1 524 -1 524

Table 6.6 (4) Radial runout of inner and outer rings

Unit: μ m

_					80		σιπε. μπι
	Nominal outs	side diameter	Radial	runout of in	ner ring of K_{ia}	assembled l	pearing
	1)	Radial	runout of o	uter ring of $K_{ m ea}$	assembled l	bearing
	mm (Over	(inch) Incl.	Class 4 Max.	Class 2 Max.	Class 3 Max.	Class 0 Max.	Class 00 Max.
	- 304.8 (14) 609.6 (24)	304.8 (14) 609.6 (24) 914.4 (36)	51 51 76	38 38 51	8 18 51	4 _ _	2 _ _
	914.4 (36)	_	76		76		_



Unit: μm

Deviati		led with a	ctive width master ou T1s		subunit	Devia		d with a n	fective wic naster inne T2s		
Clas	Shirts and Shirts and Shirts					Cla	ss 4	Cla	ss 2	Cla	ss 3
Upper						Upper	Lower	Upper	Lower	Upper	Lower
+102	0	+102	0	+102	-102	+102	0	+102	0	+102	-102
+152	-152	+102	0	+102	-102	+203	-102	+102	0	+102	-102
—	—	+178	-178 ¹⁾	+102	-102 ¹⁾	—	—	+203	-203 ¹⁾	+102	-102 ¹⁾
_	_	_	_	_	_	_	_	=	_	_	_

¹⁾ Applies to nominal bore diameters d of 406.400 mm (16 inch) or less.

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Table 6.7 Tolerance of double-row and four-row tapered roller bearings (metric series) Table 6.7 (1) Inner rings

Unit: μ m

	ninal		tion of	Variation	Variation	Radial		viation	Deviati	on of beari	ng overall	width
dian	ore neter d	diame single	bore ter in a plane Imp	of bore diameter in a single plane $V_{d\mathrm{sp}}$	of mean bore diameter $V_{d\mathrm{mp}}$	runout of inner ring of assembled bearing K_{ia}	in:	a single ner ring width $\Delta_{B{ m S}}$	Double ro	w bearing		bearing B2s
Over	Incl.	Upper	Lower	Max.	Max.	Max.	Upp	er Lower	Upper	Lower	Upper	Lower
30 50 80	50 80 120	0 0 0	-12 -15 -20	12 15 20	9 11 15	20 25 30	0 0 0	-120 -150 -200	+240 +300 +400	-240 -300 -400	- +500	_ _ _500
120 180 250	180 250 315	0 0 0	-25 -30 -35	25 30 35	19 23 26	35 50 60	0 0 0	-250 -300 -350	+500 +600 +700	-500 -600 -700	+600 +750 +900	-600 -750 -900
315 400 500	400 500 630	0 0 0	-40 -45 -60	40 45 60	30 34 40	70 80 90	0 0 0	-400 -450 -500	+800 +900 +1 000	-800 -900 -1 000	+1 000 +1 200 +1 200	-1 000 -1 200 -1 200
630 800	800 1 000	0 0	-75 -100	75 100	45 55	100 115	0 0	-750 -1 000	+1 500 +1 500	-1 500 -1 500	+1 500 +1 500	-1 500 -1 500

¹⁾ Values in dot-line frame are the **NTN** standard.

Table 6.7 (2) Outer rings

Unit: μ m

Nom			tion of	Variation	Variation	Radial	Deviation	Deviati	on of beari	ng overall	width
outs diam		diame	outside ter in a plane	of outside diameter in a single plane	of mean outside diameter	runout of outer ring of assembled	of a single outer ring width	Double ro	w bearing	Four-row	/ bearing
<i>L</i> mr		Δ_L	Omp	$V_{D\mathrm{sp}}$	V_{Dmp}	bearing K_{ea}	$\Delta C_{\rm S}$	Δ	C1s	Δι	C2s
Over	Incl.	Upper	Lower	Max.	Max.	Max.	Upper Lower	Upper	Lower	Upper	Lower
50 80 120	80 120 150	0 0 0	-16 -18 -20	16 18 20	12 14 15	25 35 40	Depends on tolerance of Δ_{Bs} in		nds on nce of n	Deper tolera Δ_{B2s} in	nce of
150 180 250	180 250 315	0 0 0	-25 -30 -35	25 30 35	19 23 26	45 50 60	relation to d of the same bearing		on to d same	relatio of the bearin	same
315 400 500	400 500 630	0 0 0	-40 -45 -50	40 45 60	30 34 38	70 80 100					
630 800 1 000	800 1 000 1 250	0 0 0	-75 -100 -125	80 100 130	55 75 90	120 140 160					
1 250	1 600	0	-160	170	100	180					

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Table 6.8 Tolerance of tapered roller bearings of J series (metric series) Table 6.8 (1) Inner rings

Non bo	re	D	eviati		mean single		diame e	eter i	n a		riation neter pla				iation ore di			Axial runout of inner ring of assembled bearing
C	l				Δ_a	/mp					V_{c}	<i>l</i> sp			V_d	mp		S_{ia}
mm		Cl	ass	CI	ass	CI	ass	Cl	ass	Class	Class	Class	Class	Class	Class	Class	Class	Class
m	m		<	1	N		С		В	K	N	С	В	K	N	С	В	В
Over	Incl.	Upper	Lower	Upper	Lower	Upper	Lower	Upper	Lower		M	ax.			Ma	ax.		Max.
10	18	0	-12	0	-12	0	-7	0	-5	12	12	4	3	9	9	5	4	3
18	30	0	-12	0	-12	0	-8	0	-6	12	12	4	3	9	9	5	4	4
30	50	0	-12	0	-12	0	-10	0	-8	12	12	4	3	9	9	5	5	4
50	80	0	-15	0	-15	0	-12	0	-9	15	15	5	3	11	11	5	5	4
80	120	0	-20	0	-20	0	-15	0	-10	20	20	5	3	15	15	5	5	5
120	180	0	-25	0	-25	0	-18	0	-13	25	25	5	3	19	19	5	7	7
180	250	0	-30	0	-30	0	-22	0	-15	30	30	6	4	23	23	5	8	8

Note: Please consult NTN Engineering for Class A bearings.

Table 6.8 (2) Outer rings

Unit: μ m

Nom outs diam	side	De	viatio	n of n	nean o single			eter	in a		neter	of out in a si ine				of m		Axial runout of outer ring of assembled bearing
L)				Δ_D	mp					V_L	Osp			V_D	mp		S_{ea}
			ass		lass	Cl	ass				Class	Class			Class	Class		
m			K		N		C		В	K	Ν	С	В	K	Ν	С	В	В
Over	Incl.	Upper	Lower	Uppe	r Lower	Upper	Lower	Uppe	r Lower		Ma	ax.			Ma	ax.		Max.
18	30	0	-12	0	-12	0	-8	0	-6	12	12	4	3	9	9	5	4	3
30	50	0	-14	0	-14	0	-9	0	-7	14	14	4	3	11	11	5	5	3
50	80	0	-16	0	-16	0	-11	0	-9	16	16	4	3	12	12	6	5	4
80	120	0	-18	0	-18	0	-13	0	-10	18	18	5	3	14	14	7	5	4
120	150	0	-20	0	-20	0	-15	0	-11	20	20	5	3	15	15	8	6	4
150	180	0	-25	0	-25	0	-18	0	-13	25	25	5	3	19	19	9	7	5
180	250	0	-30	0	-30	0	-20	0	-15	30	30	6	4	23	23	10	8	6
250	315	0	-35	0	-35	0	-25	0	-18	35	35	8	5	26	26	13	9	6
315	400	0	-40	0	-40	0	-28	0	-20	40	40	10	5	30	30	14	10	6

Note: Please consult **NTN** Engineering for Class A bearings.

Table 6.8 (3) Effective width of inner subunits and outer rings

Unit: μ m

	Nom bo diam	re						ve widt aster o			Devia				fective naster i			r ring
	a	l				Δ	T1s							Δ	T2s			
			Cla	ass	Cla	ass	Cla	ass	Cla	ass	Cla	ass	Cla	ass	Cla	ass	Cla	ass
	m	m	H	<	1	V	(2	E	3	ŀ	(1	V	(E	3
	Over	Incl.	Upper	Lower	Upper	Lower	Upper	Lower	Upper	Lower	Upper	Lower	Upper	Lower	Upper	Lower	Upper	Lower
ı	10	80	+100	0	+50	0	+100	-100	*	*	+100	0	+50	0	+100	-100	*	*
	80	120	+100	-100	+50	0	+100	-100	*	*	+100	-100	+50	0	+100	-100	*	*
	120	180	+150	-150	+50	0	+100	-100	*	*	+200	-100	+100	0	+100	-150	*	*
	180	250	+150	-150	+50	0	+100	-150	*	*	+200	-100	+100	0	+100	-150	*	*

Note: 1. " * " mark bearings are manufactured only for combined bearings. 2. Please consult **NTN** Engineering for Class A bearings.

Unit: μ m

	Deviatio	n of the	actual a	ssemble	ed bearii	ng width	1							
			Δ	Ts										
Class Class Class K N C B														
Upper	Lower	Upper	Lower	Upper	Lower	Upper	Lower							
+200 +200 +200	0 0 0	+100 +100 +100	0 0 0	+200 +200 +200	-200 -200 -200	+200 +200 +200	-200 -200 -200							
+200 +200 +350	0 -200 -250	+100 +100 +150	0 0 0	+200 +200 +350	-200 -200 -250	+200 +200 +200	-200 -200 -250							
+350	-250	+150	0	+350	-300	+200	-300							

Table 6.8 (4) Radial runout of inner and outer

$ \begin{array}{c c} \textbf{Nominal} & \textbf{Radial runout of inner ring of assembled bearing} \\ \textbf{outside} & K_{\text{ia}} \\ \textbf{diameter} & \textbf{Radial runout of outer ring of assembled bearing} \\ \end{array} $								
D		K	ea					
	Class	Class	Class	Class				
mm	K	N	С	В				
Over Incl		Max.						
18 30 30 50 50 80	18 20 25	18 20 25	5 6 6	3 3 4				
80 120 120 150 150 180	35 40 45	35 40 45	6 7 8	4 4 4				
180 250 250 315 315 400	50 60 70	50 60 70	10 11 13	5 5 5				

Note: Please consult **NTN** Engineering for Class A bearings.

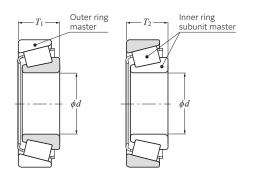


Table 6.9 Tolerance of thrust ball bearings Table 6.9 (1) Shaft washer

Unit: μm

	al bore neter				Variation diame single		Variation in thickness between shaft washer raceway and back face				
C	l		Δ_a	/mp		V_{c}	dsp		S	$S_{\mathbf{i}}$	
m	m	Class	0, 6, 5		ss 4	Class 0, 6,	5 Class 4	Class 0	Class 6	Class 5	Class 4
Over	Incl.	Upper	Lower	Upper	Lower	M	ax.		M	ax.	
— 18 30	18 30 50	0 0 0	-8 -10 -12	0 0 0	-7 -8 -10	6 8 9	5 6 8	10 10 10	5 5 6	3 3 3	2 2 2
50 80 120	80 120 180	0 0 0	-15 -20 -25	0 0 0	-12 -15 -18	11 15 19	9 11 14	10 15 15	7 8 9	4 4 5	3 3 4
180 250 315	250 315 400	0 0 0	-30 -35 -40	0 0 0	-22 -25 -30	23 26 30	17 19 23	20 25 30	10 13 15	5 7 7	4 5 5
400 500	500 630	0 0	-45 -50	0	-35 -40	34 38	26 30	30 35	18 21	9 11	6 7

Table 6.9 (2) Housing washer

Unit: μ m

	l outside neter	Deviation	on of mear in a sing		diameter	outside (tion of diameter gle plane	Variation in thickness between housing washer raceway and back face	
1	D		Δ_L	Omp		V_{I}	Osp	$S_{ m e}$	
	ım	Class	0, 6, 5		ss 4		5 Class 4	Class 0, 6, 5, 4	
Over	Incl.	Upper	Lower	Upper	Lower	M	ax.	Max.	
10 18 30	18 30 50	0 0 0	-11 -13 -16	0 0 0	-7 -8 -9	8 10 12	5 6 7		
50 80 120	80 120 180	0 0 0	-19 -22 -25	0 0 0	-11 -13 -15	14 17 19	8 10 11	Depends on tolerance of	
180 250 315	250 315 400	0 0 0	-30 -35 -40	0 0 0	-20 -25 -28	23 26 30	15 19 21	S_i against d of the same bearings	
400 500 630	500 630 800	0 0 0	-45 -50 -75	0 0 0	-33 -38 -45	34 38 55	25 29 34		

Table 6.9 (3) Bearing height Unit: μm

diam	al bore neter	bearing single-direct	of the actual g height, tion bearing 1)
Over	m Incl.	Upper	Lower
—	30	0	-75
30	50	0	-100
50	80	0	-125
80	120	0	-150
120	180	0	-175
180	250	0	-200
250	315	0	-225
315	400	0	-300
400	500	0	-350
500	630	0	-400

¹⁾ Applies to flat back face bearing of Class 0. The values are the **NTN** standard.

Table 6.10 Tolerance of thrust spherical roller bearings Table 6.10 (1) Shaft washer

Unit: μ m

Nominal bore diameter		Deviation of mean bore diameter in a single plane Δ_{dmp}		Variation of bore diameter in a single plane $V_{d\mathrm{sp}}$	Perpendicularity of shaft washer back face with respect to the bore 1)	Deviation of the actual bearing height $^{1)}$ Δ $_{Ts}$	
Over	Incl.	Upper	Lower	Max.	Max.	Upper	Lower
50 80 120	80 120 180	0 0 0	-15 -20 -25	11 15 19	25 25 30	+150 +200 +250	-150 -200 -250
180 250 315	250 315 400	0 0 0	-30 -35 -40	23 26 30	30 35 40	+300 +350 +400	-300 -350 -400
400	500	0	-45	34	45	+450	-450

¹⁾ The standard conforms to JIS B 1539.

Table 6.10 (2) Housing washer

Unit: μι

Nominal diam		outside di single	n of mean ameter in a plane
m Over	m Incl.	Upper	Lower
120	180	0	-25
180	250	0	-30
250	315	0	-35
315	400	0	-40
400	500	0	-45
500	630	0	-50
630	800	0	-75
800	1 000	0	-100

6.3 Chamfer measurements and tolerance or allowable values of tapered bore

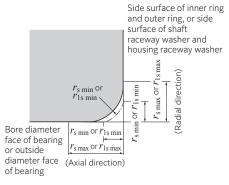
Table 6.11 Allowable critical-value of bearing chamfer Table 6.11 (1) Radial bearings (except

	tapered roller bearing) Unit: mm									
$r_{\rm S~min}^{1)}$	Nomin diam		$r_{ m s\ max}$ or	r _{1s max}						
or	(l								
$r_{1s \text{ min}}$	Over	Incl.	Radial direction	Axial direction						
0.05	_	_	0.1	0.2						
0.08	_	_	0.16	0.3						
0.1	_	_	0.2	0.4						
0.15	-	_	0.3	0.6						
0.2	_	_	0.5	0.8						
0.3	_ 40	40 —	0.6 0.8	1 1						
0.6	_ 40	40 —	1 1.3	2 2						
1	_ 50	50 —	1.5 1.9	3 3						
1.1	_ 120	120 —	2 2.5	3.5 4						
1.5	_ 120	120 —	2.3 3	4 5						
2	— 80 220	80 220 —	3 3.5 3.8	4.5 5 6						
2.1	_ 280	280 —	4 4.5	6.5 7						
2.5	— 100 280	100 280 —	3.8 4.5 5	6 6 7						
3	_ 280	280 —	5 5.5	8 8						
4	_	_	6.5	9						
5		_	8	10						
6	_	_	10	13						
7.5	_	_	12.5	17						
9.5	_	_	15	19						
12	_	_	18	24						
15	_	_	21	30						

¹⁾ These are the allowable minimum dimensions of the chamfer dimension "r" or " r_1 " and are described in the dimensional table.

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Table 6.11 (2) Tapered roller bearings of metric series Unit: mm

		metric	301103	Offic. Hilli
$r_{ m s~min}^{2)}$ or $r_{ m 1s~min}$	diame or nomin	al bore ter $d^{(3)}$ al outside eter D	$r_{ m S\ max}$ or	
0.3	_	40	0.7	1.4
	40	<u> </u>	0.9	1.6
0.6	40	_	1.3	2
1	_	50	1.6	2.5
	50	—	1.9	3
1.5	_	120	2.3	3
	120	250	2.8	3.5
	250	—	3.5	4
2	_	120	2.8	4
	120	250	3.5	4.5
	250	—	4	5
2.5	_	120	3.5	5
	120	250	4	5.5
	250	—	4.5	6
3	120 250 400	120 250 400 —	4 4.5 5 5.5	5.5 6.5 7 7.5
4		120	5	7
	120	250	5.5	7.5
	250	400	6	8
	400	—	6.5	8.5
5	_	180	6.5	8
	180	—	7.5	9
6	_	180	7.5	10
	180	—	9	11

2)These are the allowable minimum dimensions of the chamfer dimension "r" or " r_1 " and are described in the dimensional

3)Inner rings shall be in accordance with the division of "d" and outer rings with that of "D".

Note: The standard applies to the bearings whose dimensional series (refer to the dimensional table) are specified in the standard ISO 355 or JIS B 1512-3. For further information concerning bearings outside of these standards or tapered roller bearings using a US customary unit, please contact NTN Engineering.

Table 6.11 (3) Thrust bearings Unit: mm

(0)	
$r_{\rm S~min}$ or $r_{\rm 1~min}^{4)}$	$r_{ m S\ max}$ or $r_{ m 1s\ max}$ Radial and axial directions
0.05	0.1
0.08	0.16
0.1	0.2
0.15	0.3
0.2	0.5
0.3	0.8
0.6	1.5
1	2.2
1.1	2.7
1.5	3.5
2	4
2.1	4.5
3	5.5
4	6.5
5	8
6	10
7.5	12.5
9.5	15
12	18
15	21
19	25

⁴⁾ These are the allowable minimum dimensions of the chamfer dimension "r" or " r_1 " and are described in the dimensional table.

 $\phi(d+\Delta_{dmp})$

Theoretical tapered bore Tapered bore having dimensional difference of mean bore diameter within plane

Table 6.12 (1) Tolerance of tapered bores of radial bearings and tapered bores with allowable standard taper ratio 1:12 (Class 0)

	l	Jn	it٠	IIM	١

-	d mm		Δ_{dmp}		- Δ _{dmp}	$V_{d\mathrm{sp}}^{1)}$ 2)
Over	Incl.	Upper	Lower	Upper	Lower	Max.
10 18	10 18 30	+ 22 + 27 + 33	0 0 0	+ 15 + 18 + 21	0 0 0	9 11 13
30 50 80	50 80 120	+ 39 + 46 + 54	0 0 0	+ 25 + 30 + 35	0 0 0	16 19 22
120 180 250	180 250 315	+ 63 + 72 + 81	0 0 0	+ 40 + 46 + 52	0 0 0	40 46 52
315 400 500	400 500 630	+ 89 + 97 +110	0 0 0	+ 57 + 63 + 70	0 0 0	57 63 70
630 800 1 000 1 250	800 1 000 1 250 1 600	+125 +140 +165 +195	0 0 0 0	+ 80 + 90 +105 +125	0 0 0	_ _ _ _

Table 6.12 (2) Tolerance of tapered bores of radial bearings and tapered bores with allowable standard taper ratio 1:30 (Class 0)

0		Δα	/mp	Δ_{d1mp}	_ Δ _{dmp}	$V_{dsp}^{1) (2)}$
over 0	Incl.	Upper	Lower	Upper	Lower	Max.
50 80 120	80 120 180	+15 +20 +25	0 0 0	+30 +35 +40	0 0 0	19 22 40
180 250 315	250 315 400	+30 +35 +40	0 0 0	+46 +52 +57	0 0 0	46 52 57
400 500	500 630	+45 +50	0	+63 +70	0	63 70

1) Applies to all radial flat planes of tapered bores.

2) Does not apply to diameter series 7 and 8.

Note: Quantifiers

For a standard taper ratio of $\frac{1}{12}$

For a standard taper ratio of $\frac{1}{30}$, $d_1 = d + \frac{1}{30}B$

 $\Delta_{d\mathrm{mp}}$: Dimensional difference of the mean bore diameter within the flat surface at the theoretical small end of the tapered bore

 Δ_{d1mp} : Dimensional difference of the mean bore diameter within the flat surface at the theoretical large end of the tapered bore

: Unevenness of the bore diameter with the flat surface : Nominal width of inner ring

: $\frac{1}{2}$ of the tapered bore's standard taper angle

For a standard taper ratio of $\frac{1}{12}$ For standard taper ratio of $\frac{1}{30}$ $\alpha = 2^{\circ}23'9.4''$ $\alpha = 0^{\circ}57'17.4''$ = 2.38594° = 0.95484°

= 0.016665 rad = 0.041643 rad

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6.4 Bearing tolerance measurement methods

For reference, measurement methods for rolling bearing tolerances are specified in JIS B 1515-2.

Table 6.13 shows some of the major methods of measuring rotation tolerances.

Table 6.13 Rotation tolerance measurement methods

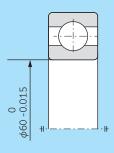
Accuracy characteristics	Measurement methods	
Radial runout of inner ring of assembled bearing (K_{ia})	Measuring load Measur	Radial runout of the inner ring is the difference between the maximum and minimum reading of the measuring device when the inner ring is turned one revolution.
Radial runout of outer ring of assembled bearing (K_{ea})	Measuring load Measuring lo	Radial runout of the outer ring is the difference between the maximum and minimum reading of the measuring device when the outer ring is turned one revolution.
Axial runout of inner ring of assembled bearing (S_{ia})	Measuring load Measuring load	Axial runout of the inner ring is the difference between the maximum and minimum reading of the measuring device when the inner ring is turned one revolution.
Axial runout of outer ring of assembled bearing $(S_{ m ea})$	Measuring load Measuring	Axial runout of the outer ring is the difference between the maximum and minimum reading of the measuring device when the outer ring is turned one revolution.
Perpendicularity of inner ring face with respect to the bore $(S_{ m d})$		The squareness of the inner ring side surface is the difference between the maximum and minimum readings of the measuring device when the inner ring is turned one revolution together with the tapered mandrel.
Perpendicularity of outer ring outside surface with respect to the face (S_{D})	$1.2r_{\rm s}$ max $1.2r_{\rm s}$ max $1.2r_{\rm s}$ Reinforcing plate	The squareness of the outer ring outside diameter surface is the difference between the maximum and minimum readings of the measuring device when the outside ring is turned one revolution along the reinforcing plate.

6.5 Geometrical product specifications (GPS)

GPS is an abbreviation of geometrical product specifications. GPS is the new drawing notation for accurately describing the geometrical specifications of product shapes, dimensions, and surface characteristics. The standard that specifies rules for making drawings with GPS is called "GPS standard."

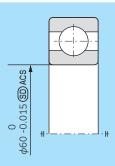
<Purpose of GPS>

While conventional drawing notation typically describes product dimensions and characteristics accurately, there are several "unclear" aspects of the conventional notation that can lead to varying interpretations (see Fig. 6.2). The main purpose of the GPS is to eliminate the ambiguity of drawing notation, thereby preventing troubles.



The dimensional tolerance of bearing bore diameters is specified in ISO/JIS as "Dimensional tolerance of mean bore diameter within plane" [see **Table 6.4 (1)**]. However, expressions such as "within plane" and "mean" are omitted in the conventional drawing notation as shown above. This can lead to several interpretations of the dimensional tolerance.

Conventional notation



GPS code "SD" represents "middle" and "ACS" represents "arbitrary cross section." By adding these two codes after the bore diameter tolerance, it is possible to clearly express "middle of the measured diameter within an arbitrary cross section", which is the dimensional tolerance of the mean bore diameter within a plane.

GPS notation

Fig. 6.2 Notation example of bearing bore diameter tolerance

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<Applying GPS to rolling bearings>

In regards to standards related to roller bearings, ISO 492 specifying the tolerance of radial bearings and ISO 199 specifying the tolerance of thrust bearings were revised with GPS in 2014. In response to this, JIS B 1514–1 and JIS B 1514–2 were also revised in 2017.

<Example of bearing drawing applying GPS>

Fig. 6.3 shows an example of a bearing drawing that uses GPS.

Drawings that use GPS include notations and codes that are different from the ones used in conventional drawings.

For details, please contact NTN Engineering.

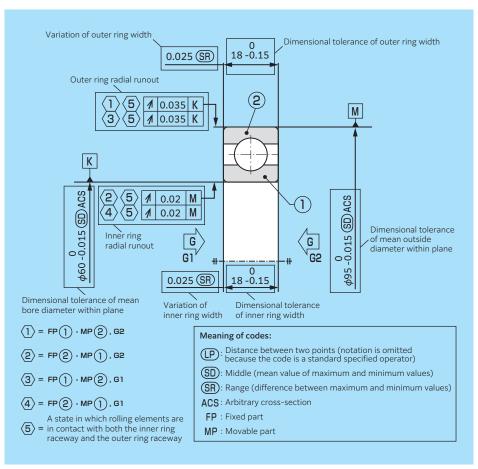


Fig. 6.3 Example of bearing drawing applying GPS